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***HIGH AND LOW SPEED DUAL DISPLACEMENT  
RADIAL PISTON MOTORS  
GD SERIES  
TECHNICAL CATALOGUE***

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***GD 100***

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***INDEX***

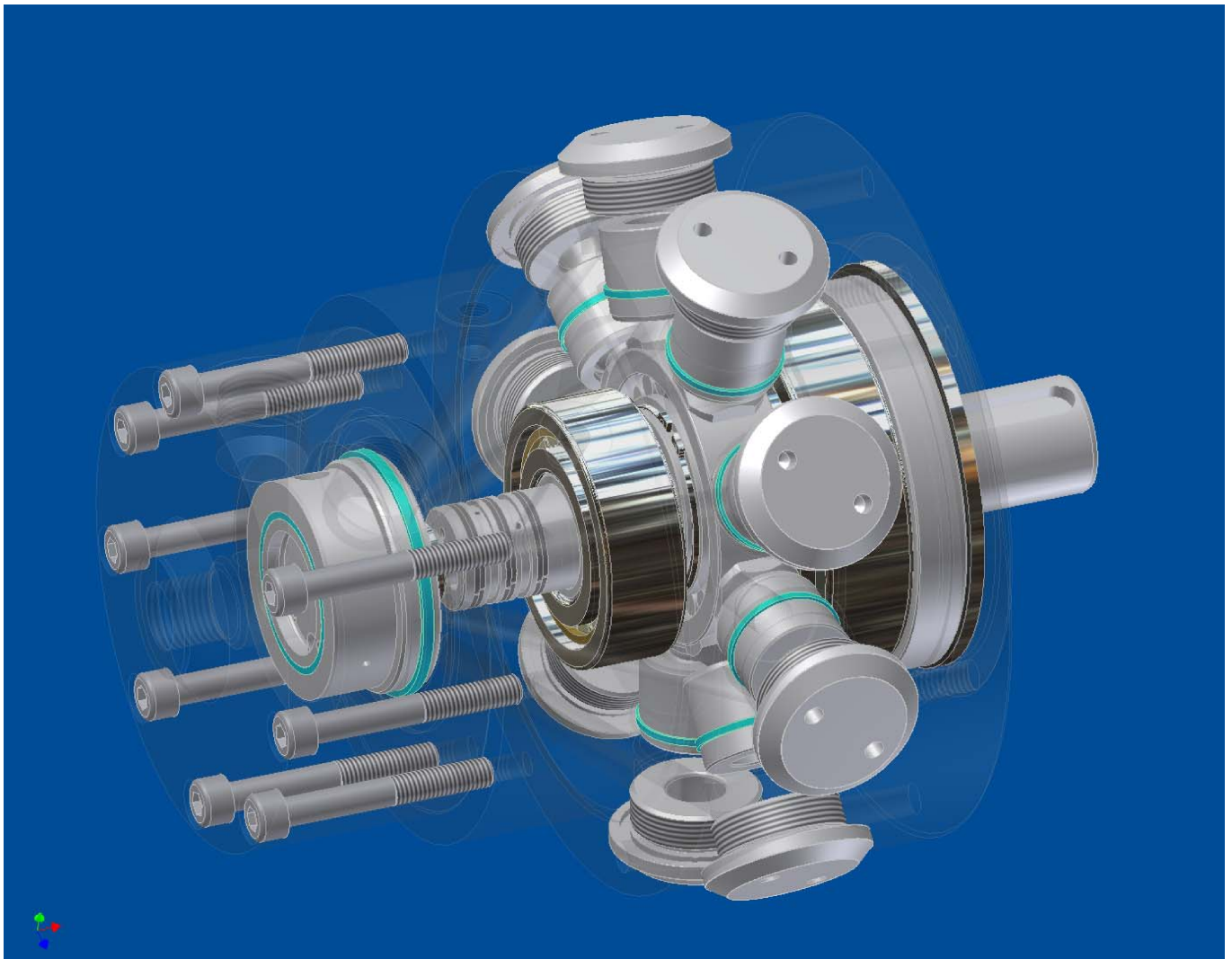
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## GENERAL INFORMATION

INTERMOT produces RADIAL PISTON HYDRAULIC MOTORS since 1985: our yearly production is more than 13.000 units which we sell all over the world through our agents and authorized sellers. Our motor range varies from 20cc to 8500cc displacement and it is completed by two-speed motors and special motors created in cooperation with our clients for different applications such as : underwater, high & low speed and wheel motors and with the possibility to assemble valves, brakes or gear reductions. You can directly contact our Technical Department which will give you all the necessary support to find the right solutions to your problems.

INTERMOT is a flexible work reality and manages deliveries also within the same day of order; we produce motors exactly interchangeable with our competitors, always ready on stock which our clients particularly appreciate.



# TECHNICAL DATA

# GD series

## DISPLACEMENT CHANGE DURING THE MOTOR FUNCTIONING

The user can choose between two displacements, acting on the hydraulic circuit. When the X port is at high pressure (system pressure) and the Y port is at low pressure (drain pressure), the motor functions at the maximum displacement, otherwise, when the Y port is at high pressure (system pressure) and the X port is at low pressure (drain pressure), the motor functions at the minimum displacement. When the X and Y ports are at low pressure the motor automatically switch in the maximum displacement.

## DISPLACEMENTS SELECTION

Not all max and minimum displacements are possible, the displacements have a range, for the maximum displacement the customer can choose between 100 and 38 cc/Rev; for the minimum displacement the user can choose between 89 and 31 cc/Rev. In the following table are showed the technical data for some of the possible displacements.

		MAX DISPLACEMENT					
Max displacement	cc/Rev	102	89	76.3	63.6	50.9	38
Specific torque	Nm/bar	1.61	1.41	1.21	1.01	0.81	0.60
Max continuous speed	Rpm	1850	1900	2100		2300	2350
Peak speed	Rpm	2200	2200	2300		2400	2400
Max freewheeling speed (*)	Rpm	2000					
Max power	Kw	60	52	44	37	31	23
	HP	82	71	59	49	42	32

		MIN DISPLACEMENT					
Min displacement	cc/Rev	89	76,3	63.6	50.9	38	31
Specific torque	Nm/bar	1.41	1.21	1.01	0.81	0.60	0.49
Max continuous speed	Rpm	1900	2100		2300	2350	2400
Peak speed	Rpm	2200	2300		2400	2400	2500
Max freewheeling speed (*)	Rpm	2000					
Max power	Kw	52	44	37	31	23	19
	HP	71	59	49	42	32	25

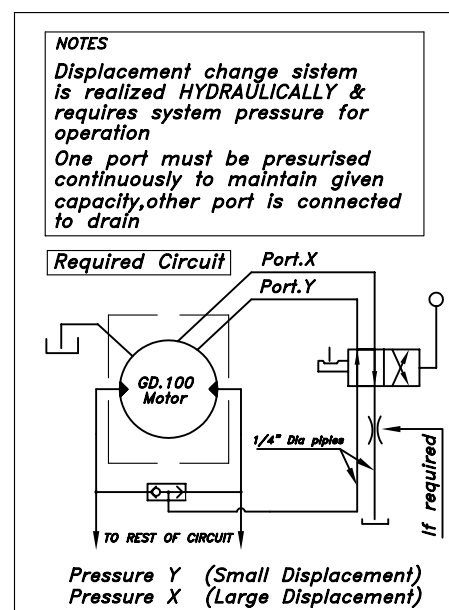
(\*) For the hydraulic circuit, please refer to page 16 (freewheeling operation).

CONT.PRESSURE	250 Bar
INT.PRESSURE	280 Bar
PEAK.PRESSURE	350 Bar
MAX.DRAIN.PRESSURE	6.0 Bar
DRY WEIGHT	30 Kg
TEMP.INTERVAL	-30 +70°C

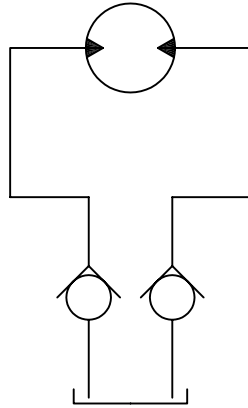
When the motor continuously works at high power values, motor flushing is needed. The recommended flushing flow is 3+5 l/min. For further information please contact Intermot technical department.

- N° of pistons: 9
- Max case pressure: 6 bar
- Max back pressure: 70 bar
- Temperature range: -30°C ÷ +70°C

For closed loop circuit applications please contact intermot technical department.



## *FREEWHEELING OPERATION*



This is the most suitable circuit for high speed freewheeling. The motor operates under vacuum conditions, therefore it can work several hours without causing any damage and overheating.

The switch from normal to freewheeling operation (and viceversa) must be done at low speed and pressure.

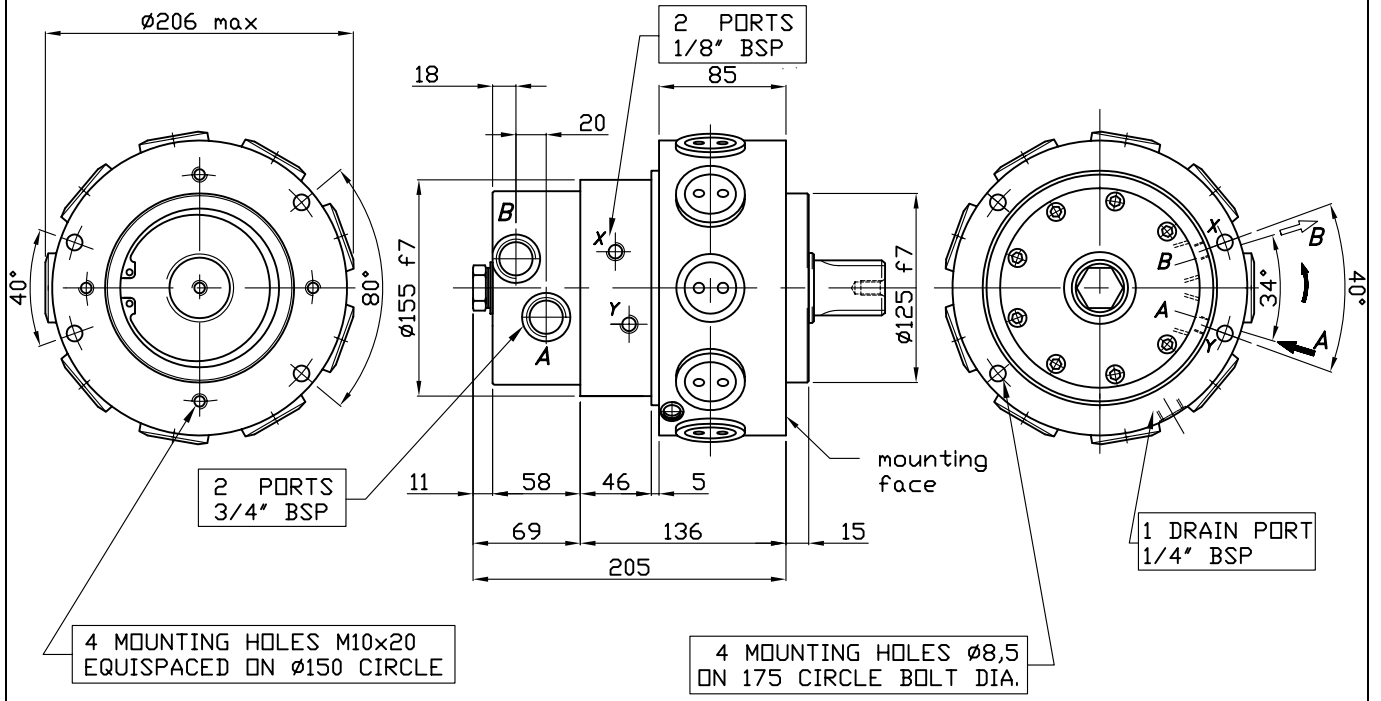
For further informations please contact Intermot technical department.

For further information contact Intermot technical department.

# SIZE

GD 100

STANDARD SERIES

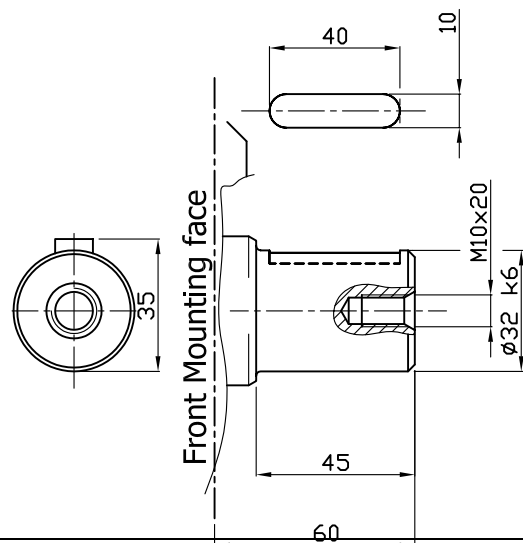
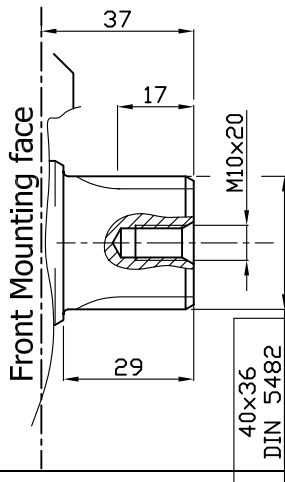


# SHAFT

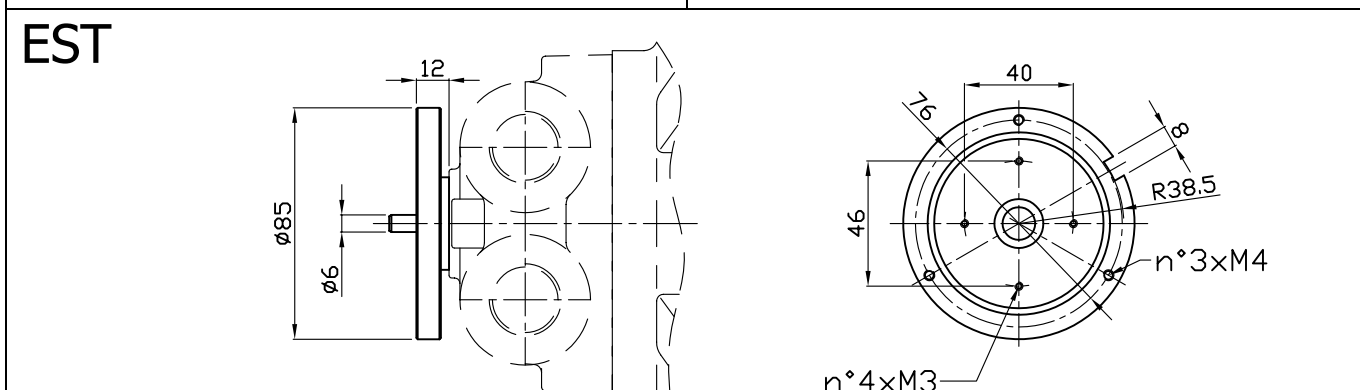
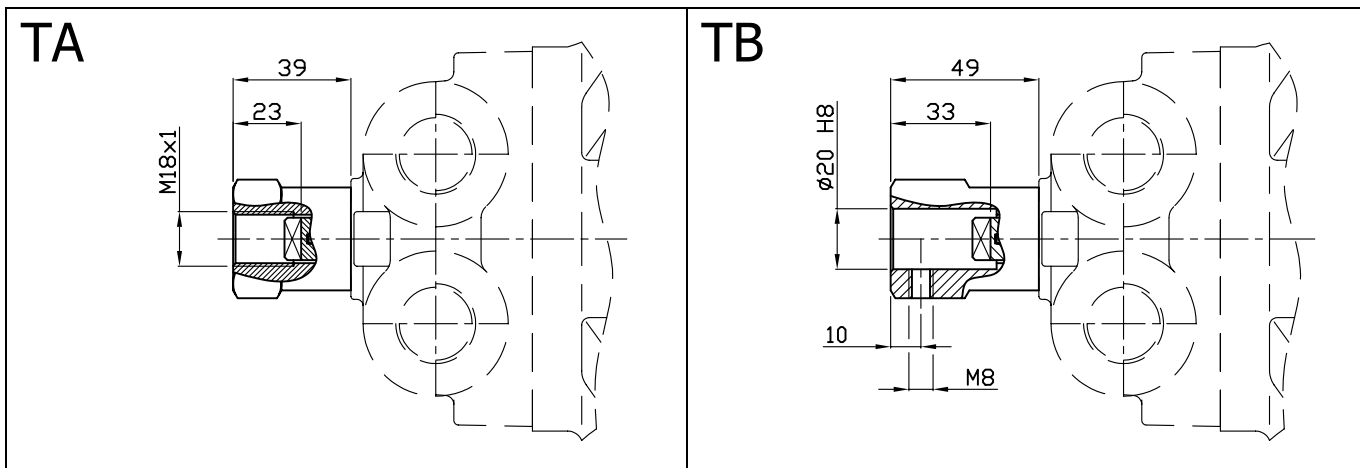
GD 100

A1: Splined Shaft

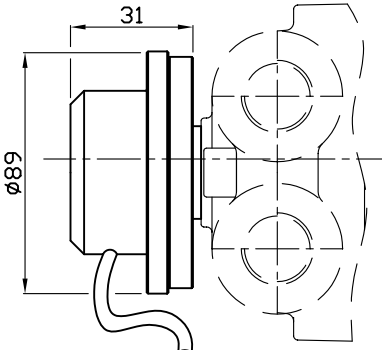
A2: Parallel shaft on request



# TACHOMETER



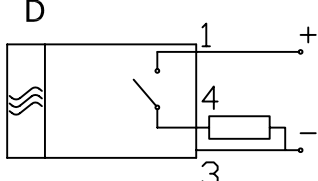
**EST.30**



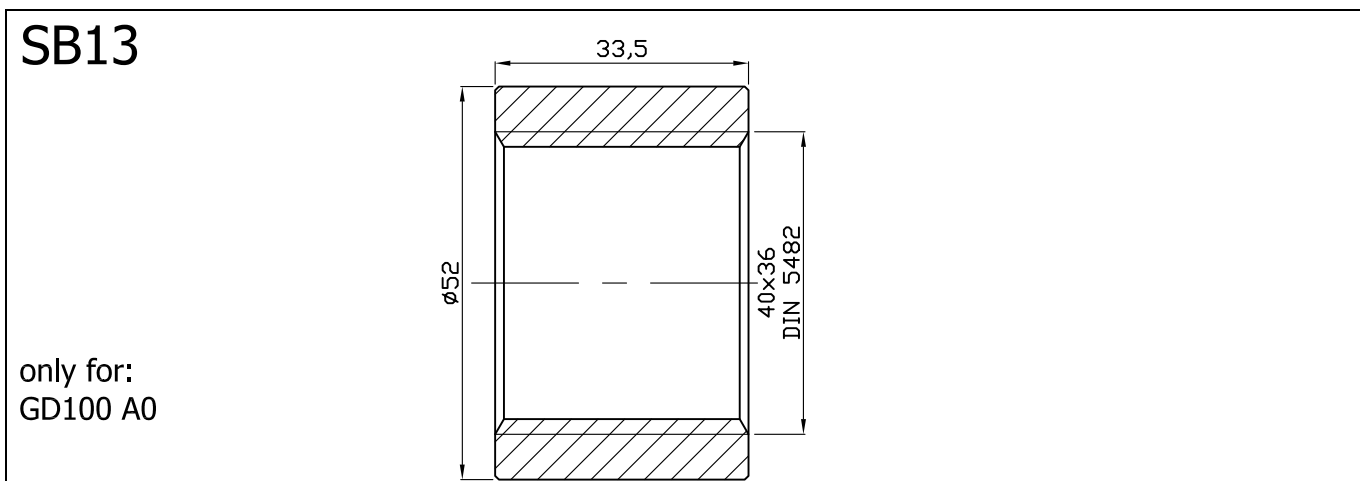
Operating parameters	E-..../3
Power supply (VDC)	10-30
Switching current (mA)	150
Frequency (Hz) 100rpm	50
Impulse/rpm	30
Operating temp. (°C)	-24/+70
Protection degree	IP67
Output	NPN
Motor type	All types

MODEL	Ø5
Torque	1 Nm

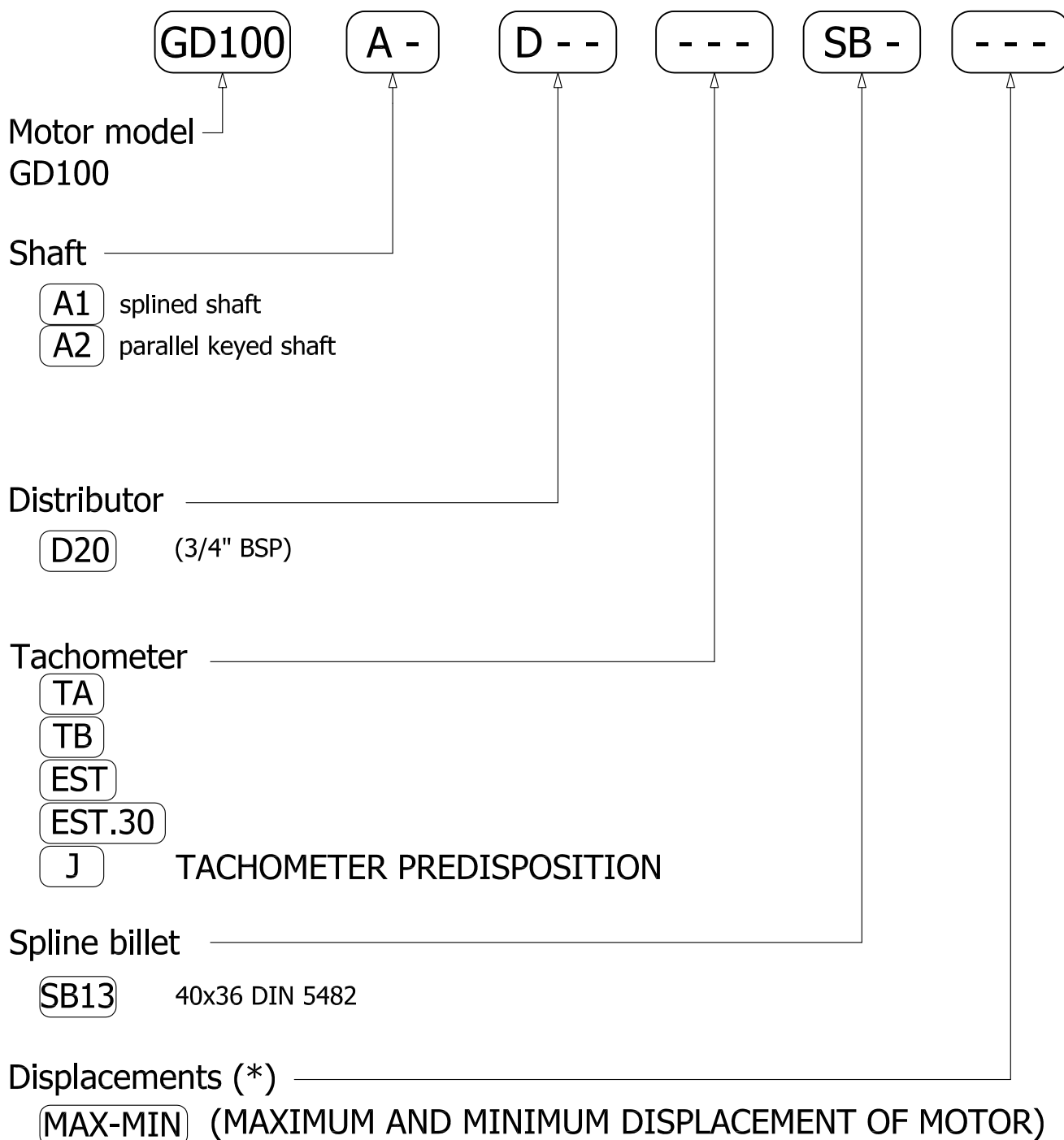
Model	Output	Fig.
E-..../.AP/...	PNP	D



# SPLINE BILLET



# ORDERING INSTRUCTIONS *GD100*



(\*) see page 15 for available maximum and minimum displacements

EXAMPLE: GD100.A1.D20.100-38  
GD100.A2.D20.TA.75-31  
GD100.A1.D20.J.60-38

# HYDRAULIC FLUIDS RECOMMENDATIONS

## HYDRAULIC FLUIDS

We recommend the use of hydraulic oils with anti-wear additives (ISO HM or HV) and minimum viscosity index of 95. Once normal working temperature is reached, oil viscosity must be at least 12 cSt, preferably in the range from 20 to 60 cSt.

Hydraulic oils meeting Denison MF-O, Vickers M-2952-S I - 286-S performance requirements and DIN 51524 specifications, are preferred.

Mineral hydraulic oils are divided into four main types, designated by the International Standards Organisation (ISO) as HH, HL, HM and HV. We advise to use only products with HM or HV specifications.

### HM type

These are the most widely employed hydraulic oils. They include small quantities of anti-wear additives to provide significant improvement in wear reduction. "Superior" quality HM type oils can be used for all equipment, with the added assurance that they will be suitable for the highest temperature.

### HV type

HV hydraulic oils show minimal change in viscosity with temperature variations.

## OIL VISCOSITY RECOMMENDATION

Room temperature HM type ISO-VG

- -20°C / 0°C BP ENERGOL HLP - HM 22
- -15°C / +5°C BP ENERGOL HLP - HM 32
- -8°C / +15°C BP BNERGOL HLP - HM 46
- 0°C / +22°C BP ENERGOL HLP - HM 68
- +8°C / +30°C BP ENERGOL HLP - HM100
- -20°C / +5°C BP BARTRAN HV 32
- -15°C / +22°C BP BARTRAN HV 46
- 0°C / +30°C BP BARTRAN HV 68

Our motors have been designed to work also with:

- oils type ATF (Automatic Transmission Fluid)
- oils with viscosity SAE 10W - 20 -30
- multigrade motor oils SAE 10 W/40 or 15 W/40
- universal oils

During cold start-up, avoid high-speed operation until the system is warmed up to provide adequate lubrication.

Continuous working temperature must not exceed 70°C.

## FIRE RESISTANT OIL LIMITATIONS

	Max cont. pressure	Max int. pressure	Max speed
HFA, 5-95% oil-water	103	138	50%
HFB, 60-40% oil-water	138	172	100%
HFC, water-glycol	103	138	50%
HFD, ester phosphate	250	293	100%

## FILTRATION

Hydraulic systems oil must always be filtered.

The choice of filtration grade derives from needs of service life and money spent. In order to obtain stated service life it is important to follow our recommendations concerning filtration grade.

When choosing the filter it is important to consider the amount of dirt particles that filter can absorb and still operate satisfactorily. For that reason we recommend filters showing when you need to substitute filtering cartridge.

- 25 µm filtration required in most applications
- 10 µm filtration in closed circuit applications

## OXIDATION

Hydraulic oil oxidizes with time of use and temperature. Oxidation causes changes in colour and smell, acidity increase or sludge formation in the tank. Oxidation rate increases rapidly at surface temperatures above 60°C, in these situations oil should be checked more often.

The oxidation process increases the acidity of the fluid; the acidity is stated in terms of the "neutralization number". Oxidation is usually slow at the beginning and then it increases rapidly.

A sharp increase (by a factor of 2 to 3) in neutralization number between inspections shows that oil has oxidized too much and should be replaced immediately.

## WATER CONTENT

Oil contamination by water can be detected by sampling from the bottom of the tank. Most hydraulic oils repel the water, which then collects at the bottom of the tank. This water must be drained off at regular intervals. Certain types of transmission oils and engine oils emulsify the water; this can be detected by coatings on filter cartridges or a change in the colour of the oil. In such cases, obtain your oil supplier advice.

## DEGREE OF CONTAMINATION

Heavy contamination of the oil causes wear rising in hydraulic system components. Contamination causes must be immediately investigated and remedied.

## ANALYSIS

It is recommended oil being analyzed every 6 months. The analysis should cover viscosity, oxidation, water content, additives and contamination. Most oil suppliers are equipped to analyze oil state and to recommend appropriate action. Oil must be immediately replaced if the analysis shows that it is exhausted.



## INSTRUCTIONS AND ADVICES

### INSTALLATION

Hoses and piping must be clean and free from contamination. No other special requirements are necessary.

- Motor can be mounted in any position
- In run-away conditions you must use counterbalance valves
- Consult factory for intermittent applications

Splined adaptors (sleeves) are available upon request.

### INSTALLATION CIRCUIT

The choice of open or closed loop circuit will be determined by the application.

Open loop circuits are cheaper and simpler to install.

Closed loop circuit is a superior circuit and usually takes up less space. It also offers better control features.

### START UP

Motor case and pistons must be completely filled with oil before starting.

Do not load motor to maximum working pressure. Increase load gradually at start-up.

### CASE DRAIN – CASE PRESSURE

Connect the case drain directly to tank.

The case drain port on the motor must be located on the highest point of the installation to ensure that the motor will always be full of oil. The case drain pressure must not exceed 6 bar continuous pressure.

### IMPORTANT

When the motor is installed vertically with shaft pointing upwards, consult our Technical Department. If the motor is connected to high inertial loads, the hydraulic system must be designed to prevent peaks of pressure and cavitation.

### TEMPERATURE

Maximum oil temperature must not exceed 70°C. Heat exchangers must be used with higher temperatures.

### VISCOSITY

The motor works satisfactory in a range of 3°E to 10°E oil viscosity. Best performance is obtained at the highest viscosity.

### BACK PRESSURE

Don't exceed 70 bar back pressure.

### HIGH PEAKS APPLICATIONS

In case of high pressure peaks applications, a Nitemper treatment on motor body is suggested to increase wear and tear resistance.

### CONTINUOUS HIGH SPEED DUTY

In case of continuous high speed duty, it is suggested to mount a central reinforced bearing on motor shaft, please contact our Technical Department.

### MINIMUM SPEED

Standard minimum speed is about 5 to 40 rpm (depending on motor displacement). If you need less speed, it is possible to modify some parts of the distributor.

### FLUSHING

In the need of Flushing, a 2nd drain hole is available upon request. When flushing is not available, it is possible to create an inner motor drain to help cooling.

### COOLING FLOW

If the motor operates in the Intermittent Power zone, it may require a cooling flow of 20 l/min (5 gpm) to keep a drain flow viscosity of 40 cSt minimum.

**FOR MORE DETAILS ON THE ABOVE MENTIONED ARGUMENTS AND FOR ANY FURTHER INFORMATION PLEASE CONTACT OUR TECHNICAL DEPARTMENT.**

## BEARINGS

Bearings lifetime depends on the type of bearing, on motor speed and on working loads.

Lifetime is measured by  $L_{10}$  which is called "theoretic lifetime". It represents the number of cycles that 90% of identical bearings can effort at the same load without showing wear and tear. It is calculated by the following equation:

$$L_{10} = \left( \frac{C}{P} \right)^p$$

where: C = theoretical dynamic coefficient (depending on the bearing size)

P = radial load

p = exponent (p=3 for ball bearings, p=10/3 for roller bearings)

When you work at constant speed, you can calculate the lifetime in hours with the following equation:

$$L_{10h} = \frac{10^6 \cdot L_{10}}{60 \cdot \text{rpm}} = \frac{10^6}{60 \cdot \text{rpm}} \left( \frac{C}{P} \right)^p \text{ [h]}$$

When you don't have only radial or axial loads, you have to calculate an equivalent load:

$$P = X \cdot F_R + Y \cdot F_A$$

Where

$F_R$  = radial load,

X = radial coefficient,

$F_A$  = axial load,

Y = axial coefficient

While  $F_R$  and  $F_A$  come from working conditions (i.e. torque),

X and Y depend on the type of bearing and on the ratio  $\frac{F_A}{F_R}$ .

To help you in the expected lifetime calculation, Intermot provides you with an EXCEL calculation sheet. With this instrument you can easily calculate lifetime: you only need to choose the motor model, put speed, pressure and loads.

**For further information or to have the calculation sheet, please contact our Technical Department.**

# FLUSHING

## FLUSHING FLOW

Cooling flow is necessary to assure the minimum oil viscosity and depends on motor displacement.

Motor	Flushing flow [l/min]
GD 100	3÷5

## FLUSHING IN PERFORMANCE DIAGRAMS

Each performance diagram shows working conditions where flushing is suggested (areas numbered from 4 to 6 in each performance diagram).

Area 1: Continuous operation

Area 2: Intermittent operation for period 3-5 minute every 10-15 minute

Area 3: Intermittent operation for very short period (3-5 seconds every 10-15 minutes)

Area 4: **Continuous operation with flushing**

Area 5: **Intermittent operation for period 3-5 minute every 10-15 minute with flushing**

Area 6: **Intermittent operation for very short period (3-5 seconds every 10-15 minutes) with flushing**

## HIGH VOLUMETRIC EFFICIENCY MOTORS

On radial piston hydraulic motors with high volumetric efficiency, and therefore Intermot G series, there can be a phenomenon of oil-overheating in the body motor.

Oil drawing from the piston and from the distributor goes into body motor. When this oil quantity is very scanty, it means there's a good volumetric efficiency. In some cases this is positive, like for winch on crane truck or trawl winch, because high volumetric efficiency avoids motor rotation even under external stress.

This scanty quantity of oil is not a problem because the motor works at high pressure only for a short period of time.

In other cases, this high efficiency can cause problems on the motor because oil exchange is missing.

In fixed applications, for example, where the motor is running constantly for 8 or more hours a day (like injection machines for plastic materials, press, bending machines, etc.) high volumetric efficiency can create temperature increasing in motor body.

In this case temperature increasing is to be avoided with the use of flushing.

Flushing consists in carrying fresh oil (taken from hydraulic circuit) in the body motor.

Oil is usually taken from return line to avoid any loss of efficiency.

In this way, all internal parts of the motor are protected with this lubrication and cooled with fresh oil, so that total efficiency is optimised.

## ***DRAIN RECOMMENDATIONS***

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### **IMPORTANT**

For all motors GD series, it is necessary TO  
FILL the motor case with hydraulic fluid,  
through the drain pipe, before start-up.

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## ***PERFORMANCES***

For obtain the diagrams in wich are showed the motor characteristics contact Intermot technical departement.

# SHAFT SEAL FEATURES

Type: BABSL  
 Form: AS DIN 3760  
 Material: SIMRIT® 72 NBR 902  
 SIMRIT® 75 FKM 595

## 1. Features

SIMMERRING® radial shaft seal with rubber covered O.D., short, flexibility suspended, spring loaded sealing lip and additional dust lip: see Part B/ SIMMERRING®, sections 1.1 and 2.

## 2. Material

Sealing lip and O.D.:

- Acrylonitrile-butadiene rubber with 72 Shore

A hardness (designation: SIMRIT® 72 NBR 902)

- Fluoro rubber with 75 Shore A hardness (designation: SIMRIT®75 FKM 595)

Metal insert:

- Plain steel DIN 1624

Spring:

- Spring steel DIN 17223

## 3. Application

For sealing pressurised media without additional backup ring, e. g. for rotational pressure sealing in hydraulic pumps, hydraulic motors, hydrodynamic clutches. Rubber covered O.D. assures sealing in the housing bore even in case of considerable surface roughness, thermal expansion or split housing.

Particularly suitable for sealing low viscosity and gaseous media.

Where high thermal stability and chemical resistance are required, SIMRIT® 75 FKM 595 material should be used.

Additional dust lip to avoid the entry of light and medium dust and dirt.

## 4. Operating conditions

See Part B/ SIMMERRING®, sections 2. 4.

Media: mineral oils, synthetic oils

Temperature: -40°C to +100°C (SIMRIT® 72 NBR 902)  
 -40°C to +160°C (SIMRIT® 75 FKM 595)

Surface speed: up to 5 m/s

Working pressure: see diagram 1

Maximum permitted values, depending on other operating conditions.

## 5. Housing and Machining Criteria

See Part B/ SIMMERRING®, sections 2.

Shaft:	Tolerance:	ISO h11
	Concentricity:	IT 8
	Roughness:	Ra=0.2-0.8 µm Rz=1-4 µm Rmax=6 µm
	Hardness:	45-60 HRc
	Roughness:	non oriented; preferably by plunge grinding
Housing:	Tolerance:	ISO H8
	Roughness:	Rmax<25 µm

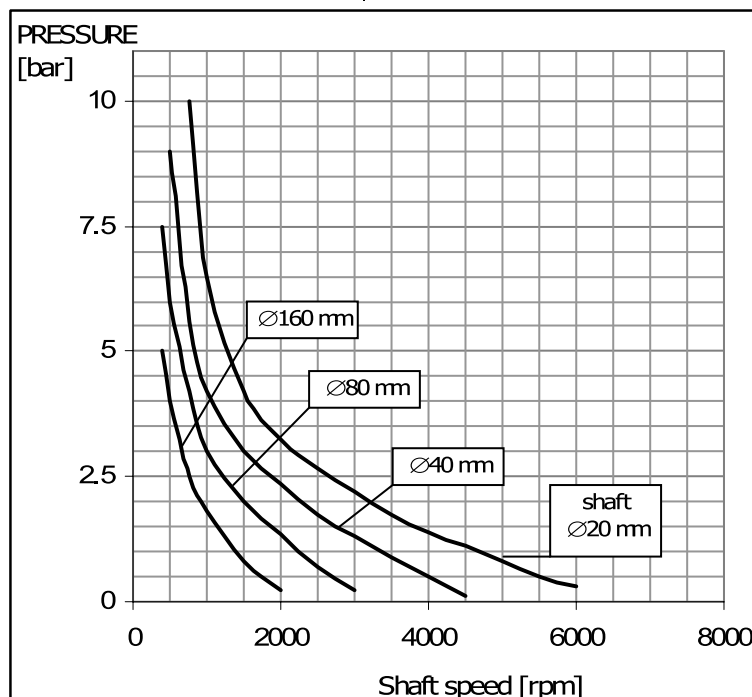


Diagram 1: Pressure Loading Limits

For more details please contact our Technical Department.

## FORMULAS

• TORQUE (1)	Torque = (specific torque) · (pressure)
• TORQUE (2)	Torque [Nm] = $\frac{\text{displacement [cc/rev]} \cdot \text{pressure [bar]}}{62.8}$
• POWER (1)	Power [kW] = $\frac{\text{Torque [Nm]} \cdot \text{speed [rpm]}}{9549}$
• POWER (2)	Power [CV] = $\frac{\text{Torque [Nm]} \cdot \text{speed [rpm]}}{7023}$
• SPEED	speed [rpm] = $\frac{\text{flow rate [l/min]} \cdot 1000}{\text{displacement [cc/rev]}}$
• REQUIRED MOTOR DISPLACEMENT	displacement [cc/rev] = $\frac{\text{max required torque [Nm]} \cdot 62.8}{\text{max pressure [bar]}}$
• REQUIRED PUMP FLOW RATE	flow [l/min] = $\frac{\text{displacement [cc/rev]} \cdot \text{max speed [rpm]}}{1000}$

## CONVERSIONS

<b>LENGTH</b>	1 m	=	39.3701 in	<b>FORCE</b>	1 N	=	0.102 kgf
		=	3.2808 ft			=	0.2248 lbf
		=	1.0936 yd		1 kgf	=	2.205 lbf
		=	1000 mm			=	9.806 N
	1 in	=	0.0833 ft		1 lbf	=	0.4536 kgf
		=	25.4 mm			=	4.448 N
	1 ft	=	0.3048 m	<b>PRESSURE</b>	1 bar	=	14.223 psi
		=	0.3333 yd			=	0.99 atm
		=	12 in			=	1.02 ata
	1 yd	=	0.9144 m			=	100000 Pa
		=	3 ft			=	100 kPa
		=	36 in			=	0.1 MPa
	1 km	=	1000 m		1 psi	=	0.0703 bar
		=	1093.6 yd	<b>FLOW</b>	1 l/min	=	0.264 gpm
		=	0.6214 mile			=	1000 cc/min
	1 mile	=	1.609 km		1 gpm	=	3.785 l/min
		=	1760 yd			=	3785 cc/min
<b>MASS</b>	1 kg	=	2.2046 lb		1 m <sup>3</sup> /s	=	60000 l/min
	1 lb	=	0.4536 kg			=	15852 gpm
<b>SPEED</b>	1 m/s	=	3.6 km/h	<b>POWER</b>	1 kW	=	1.341 HP
		=	2.237 mph			=	1.3596 CV
		=	3.2808 ft/s		1 HP	=	0.7457 Kw
	1 km/h	=	0.2778 m/s			=	1.0139 CV
		=	0.6214 mph	<b>TORQUE</b>	1 Nm	=	0.102 kgm
		=	0.9113 ft/s			=	0.7376 lbf ft
	1 mph	=	1.609 km/h		1 kgm	=	9.806 Nm
		=	0.447 m/s			=	7.2325 lbf ft
		=	1.467 ft/s		1 lbf ft	=	0.1383 kgm
	1 ft/s	=	0.3048 m/s			=	1.3558 Nm
		=	1.0973 km/h				
		=	0.6818 mph				